# Lecture 18 – Cycles Purdue ME 200, Thermodynamics I

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# Outline

Cycles

Example #1

Example #2

# Vocabulary reminders

- the state of a system determines all its properties
- a process is a transition between two equilibrium states
- a cycle is a sequence of processes that ends in the initial state
- $\implies$  at the end of a cycle, all properties return to their initial values

## 1st law for cycles

- for any closed system undergoing any process,  $\Delta E = Q W$ 
  - $\diamond E$  is the system's total energy
  - $\diamond \ Q$  is the net heat transfer into the system
  - $\diamond$  W is the net work done by the system
- since E is a property,  $\Delta E = 0$  over any cycle
- so for cycles, 1st law reduces to  ${\it Q}={\it W}$

## Sign conventions, revisited

- so far, we've defined
  - $\diamond~$  heat transfer into the system as positive
  - $\diamond~$  work done by the system as positive
- we scrap that convention for cycles (sorry)
- new convention:

arrows point in the direction of positive energy flow

• with this convention, 1st law for cycles becomes

$$\sum \mathsf{energy} \ \mathsf{inflows} = \sum \mathsf{energy} \ \mathsf{outflows}$$

• to distinguish inflows from outflows, check the picture

### Power cycles



• 1st law: 
$$Q_{in} = W + Q_{out}$$

# Refrigeration and heat pump cycles



• 1st law: 
$$Q_{\sf in} + W = Q_{\sf out}$$

# Efficiencies and coefficients of performance (COPs)

• we quantify cycle performance through benefit-to-cost ratios:

 $\label{eq:performance} \text{performance metric} = \frac{\text{beneficial thing}}{\text{costly thing}}$ 

• the metric, benefit and cost definitions vary between cycles

cycle	metric	benefit	cost
power	efficiency $\eta$	W	$Q_{in}$
refrigeration	COP $\beta$	$Q_{in}$	W
heat pump	COP $\gamma$	$Q_{\rm out}$	W

# Efficiencies and COPs (continued)

• for power cycles,

$$\eta = \frac{W}{Q_{\rm in}} = \frac{Q_{\rm in} - Q_{\rm out}}{Q_{\rm in}} = 1 - \frac{Q_{\rm out}}{Q_{\rm in}}$$

• for refrigeration cycles,

$$\beta = \frac{Q_{\rm in}}{W} = \frac{Q_{\rm in}}{Q_{\rm out} - Q_{\rm in}}$$

• for heat pump cycles,

$$\gamma = rac{\mathcal{Q}_{\mathsf{out}}}{\mathcal{W}} = rac{\mathcal{Q}_{\mathsf{out}}}{\mathcal{Q}_{\mathsf{out}} - \mathcal{Q}_{\mathsf{in}}}$$

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Cycles

### Example #1

Example #2

A refrigerator steadily draws 200 W of electric power and rejects energy by heat transfer to the kitchen air at a rate of 800 W. Determine the rate of heat transfer from the refrigerated space and the refrigerator's COP.

# System diagram



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## Given and find

- given:
- find:
  - $\begin{array}{c} \diamond \quad \dot{Q}_{\rm in} \\ \diamond \quad \beta \end{array}$

# Assumptions and basic equations

#### • assume:

 $\diamond~$  steady, cyclic operation

### • basic equations:

- $\diamond~$  1st law for refrigeration cycles:  $\mathit{Q}_{\mathsf{in}} + \mathit{W} = \mathit{Q}_{\mathsf{out}}$
- $\diamond$  refrigerator COP:  $\beta = Q_{in}/W$

### Solution

- call the time to complete one cycle  $\Delta t$
- then the rate of heat transfer from the refrigerated space is

$$\dot{Q}_{
m in} = rac{Q_{
m in}}{\Delta t} = rac{Q_{
m out} - W}{\Delta t}$$

• but  $Q_{ ext{out}} = \dot{Q}_{ ext{out}} \Delta t$  and  $W = \dot{W} \Delta t$ , so

$$\dot{Q}_{\rm in} = \frac{\dot{Q}_{\rm out}\Delta t - \dot{W}\Delta t}{\Delta t} = \dot{Q}_{\rm out} - \dot{W}$$
$$= 800W - 200W = 600W$$

# Solution (continued)

• similarly, the refrigerator's COP is

$$\beta = \frac{Q_{\text{in}}}{W} = \frac{\dot{Q}_{\text{in}}\Delta t}{\dot{W}\Delta t} = \frac{\dot{Q}_{\text{in}}}{\dot{W}}$$
$$= \frac{600W}{200W} = 3$$

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### Problem statement

Viewing the steady vapor-compression cycle pictured below as (a) a refrigerator and (b) a heat pump, express the COPs in terms of the specific enthalpies  $h_1$ ,  $h_2$  and  $h_3$ .



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## Given and find

- given:
  - $\diamond$   $h_1$ ,  $h_2$ ,  $h_3$
- find:

$$\label{eq:basic} \begin{array}{l} \diamond \hspace{0.2cm} \beta = \dot{Q}_{\rm in} / \dot{W} \\ \diamond \hspace{0.2cm} \gamma = \dot{Q}_{\rm out} / \dot{W} \end{array} \end{array}$$

## Assumptions and basic equations

#### • assume:

- $\diamond~$  steady cyclic operation
- $\diamond~$  no changes in KE or PE
- $\diamond\,$  no heat transfer across the compressor boundary
- basic equations: 1st law in rate form for the
  - $\diamond$  compressor:  $\dot{m}h_1 + \dot{W} = \dot{m}h_2$
  - $\diamond$  condenser:  $\dot{m}h_2 = \dot{Q}_{out} + \dot{m}h_3$
  - ♦ expansion valve:  $\dot{m}h_3 = \dot{m}h_4$
  - $\diamond$  evaporator:  $\dot{m}h_4 + \dot{Q}_{in} = \dot{m}h_1$
- $\star$  sign convention: energy flow is + in direction of  $\rightarrow$

# Solution to part (a)

• from the compressor energy balance,

$$\dot{W}=\dot{m}(h_2-h_1)$$

• from the evaporator energy balance,

$$\dot{Q}_{
m in}=\dot{m}(h_1-h_4)$$

• from the expansion valve energy balance,  $h_4 = h_3$ , so

$$\beta = \frac{\dot{Q}_{\text{in}}}{\dot{W}} = \frac{\dot{m}(h_1 - h_4)}{\dot{m}(h_2 - h_1)} = \frac{h_1 - h_4}{h_2 - h_1} = \frac{h_1 - h_3}{h_2 - h_1}$$

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# Solution to part (b)

• from the compressor energy balance,

$$\dot{W}=\dot{m}(h_2-h_1)$$

• from the condenser energy balance,

$$\dot{Q}_{
m out}=\dot{m}(h_2-h_3)$$

SO

$$\gamma = \frac{\dot{Q}_{\text{out}}}{\dot{W}} = \frac{\dot{m}(h_2 - h_3)}{\dot{m}(h_2 - h_1)} = \frac{h_2 - h_3}{h_2 - h_1}$$