Thermal storage and water heaters

Purdue ME 597, Distributed Energy Resources

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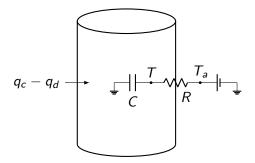
Three types of thermal storage

Domestic hot water use

Resistance water heaters

Heat-pump water heaters

Lumped sensible thermal storage



- C (kWh/°C) is thermal capacitance of storage medium
- R (°C/kW) is thermal resistance of tank wall
- q_c (kW) is charging thermal power
- q_d (kW) is discharging thermal power
- assumption: temperature is spatially uniform inside tank

Lumped sensible thermal storage (continued)

- suppose tank temperature satisfies $T \in [\underline{T}, \overline{T}]$
- choose an arbitrary reference temperature ('datum') T_{LS}
- define stored energy x (kWh) as zero when $T = T_{LS}$:

$$x = C(T - T_{LS}) \in [C(\underline{T} - T_{LS}), C(\overline{T} - T_{LS})]$$

• from conservation of energy,

$$\frac{\mathrm{d}x}{\mathrm{d}t} = \frac{T_a - T}{R} + q_c - q_d$$
$$= \frac{T_a - (T_{\mathrm{LS}} + x/C)}{R} + q_c - q_d$$
$$= -\frac{x}{RC} + q_c + w, \text{ where } w = \frac{T_a - T_{\mathrm{LS}}}{R} - q_d$$

Lumped sensible thermal storage in a 'box of rocks'



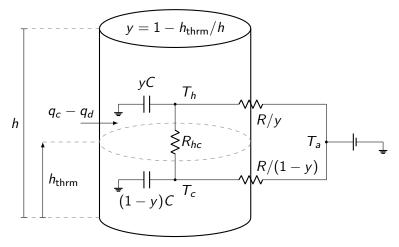
Green Energy Times: Electric Thermal Storage

Lumped sensible thermal storage in a 'box of rocks'



Steffes: Comfort Plus Forced Air

Stratified sensible thermal storage



- hot column sits on cold column, separated by thermocline
- T_h and T_c are constant; charging moves thermocline down

Stratified sensible thermal storage (continued)

• define column energies relative to (arbitrary) T_{SS} :

$$x_h = yC(T_h - T_{SS}), \ x_c = (1 - y)C(T_c - T_{SS})$$

• then total energy is

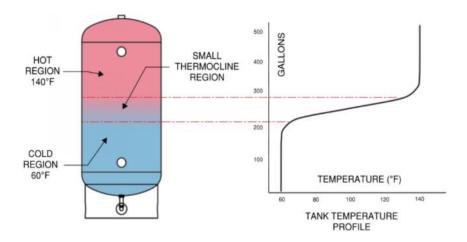
$$x = x_h + x_c = \dots = C[y(T_h - T_c) + T_c - T_{SS}]$$

$$\in [C(T_c - T_{SS}), C(T_h - T_{SS})]$$

• from conservation of energy,

$$\frac{\mathrm{d}x}{\mathrm{d}t} = \frac{(T_a - T_h)y}{R} + \frac{(T_a - T_c)(1 - y)}{R} + q_c - q_d$$
$$= \dots = -\frac{x}{RC} + q_c + w \text{ where } w = \frac{T_a - T_{\mathrm{SS}}}{R} - q_d$$

Stratified sensible thermal storage in hot water



Bonneville Power Administration (2022): Improving thermal energy storage to reduce installation costs for heat pump water heating systems

Latent thermal storage

- tank contains M kg of material that freezes and melts
- latent heat of fusion L (kWh/kg), melting point T_m (°C)
- liquid mass $m_\ell \in [0, M]$ (kg) increases as tank charges
- define energy relative to (arbitrary) T_L :

$$x = C(T_m - T_L) + Lm_{\ell} \in [C(T_m - T_L), C(T_m - T_L) + LM]$$

• from conservation of energy,

$$\frac{\mathrm{d}x}{\mathrm{d}t} = q_c + w$$
 where $w = \frac{T_a - T_m}{R} - q_d$

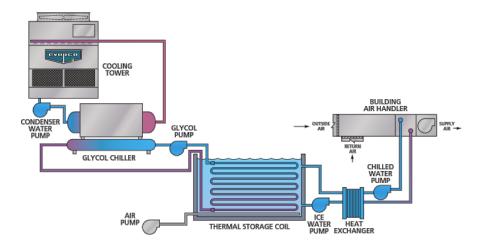
Latent thermal storage in ice





Evapco: Extra-Pak Ice Coils

Latent thermal storage in ice



Evapco: Extra-Pak Ice Coils

Summary

• all three storage types follow 'heat battery' model:

$$\frac{\mathsf{d}x}{\mathsf{d}t} = -\frac{x}{\tau} + q_c + w, \ \underline{x} \le x \le \overline{x}$$

• in discrete time with $a = e^{-\Delta t/\tau}$,

$$x(k+1) = \begin{cases} ax(k) + (1-a)\tau(q_c(k) + w(k)) & \text{if } \tau < \infty \\ x(k) + \Delta t(q_c(k) + w(k)) & \text{if } \tau = \infty \end{cases}$$

	Lumped sensible	Stratified sensible	Latent
X	$C(T - T_{LS})$	$C[y(T_h - T_c) + T_c - T_{SS}]$	$C(T_m - T_L) + Lm_\ell$
\overline{X}	$C(\overline{T} - T_{LS})$	$C(T_h - T_{SS})$	$C(T_m - T_L) + LM$
<u>x</u>	$C(\underline{T} - T_{LS})$	$C(T_c - T_{SS})$	$C(T_m - T_L)$
au	RC	RC	∞
W	$(T_a - T_{\rm LS})/R - q_d$	$(T_a - T_{\rm SS})/R - q_d$	$(T_a - T_m)/R - q_d$

One model can capture switches between storage types

- $\bullet\,$ example: suppose tank goes stratified \rightarrow lumped as it heats
- goal: choose T_{LS} , T_{SS} , <u>T</u> so model switches smoothly
- to get $w_{LS} = w_{SS}$, need $T_{LS} = T_{SS}$ (might as well choose $T_{LS} = T_{SS} = T_a$ so $w_{LS} = w_{SS} = -q_d$)

• to get
$$\underline{x}_{LS} = \overline{x}_{SS}$$
, need $\underline{T} = T_h$

• with these choices, one model captures full operating range:

$$\begin{aligned} \frac{\mathrm{d}x}{\mathrm{d}t} &= -\frac{x}{\tau} + q_c - q_d\\ C(T_c - T_a) &\leq x \leq C(\overline{T} - T_a) \end{aligned}$$

• stratified \leftrightarrow lumped switch happens when $x = C(T_h - T_a)$

Charging and discharging efficiencies

- discharging typically takes little energy (just a pump or fan)
- charging typically uses a heat pump or resistance heat
- heat pump COPs depend on tank and ambient temperatures
- for resistance, $\mathsf{COP}\approx 1$

Typical parameter values

- thermal capacitance is $C = \rho c V$
- thermal resistance is R = 1/(UA)
- typically, $U \approx 0.0005$ to 0.001 kW/(°C m²)
- for liquid water at 50 $^\circ\text{C}\textsc{,}$

$$◇$$
 ρ = 988 kg/m³
 $◇$ c = 4.17 kJ/(°C kg) = 0.00116 kWh/(°C kg)
 $◇$ ρc = 1.15 kWh/(°C m³)
 $◇$ L = 334 kJ/kg = 0.09 kWh/kg

• for a cylinder with radius r and height h,

$$\diamond V = \pi r^2 h$$

$$\diamond A = 2\pi rh + 2\pi r^2 = 2V(1/r + 1/h)$$

• stratified sensible thermal storage capacity:

$$\overline{x} - \underline{x} = C(T_h - T_c)$$

- typical $V \approx 0.2$ to 0.3 m³, so $C \approx 0.25$ to 0.35 kWh/°C
- typically, $T_c pprox 15~^\circ {
 m C}$
- below ~49 $^{\circ}\text{C}$, legionella bacteria can grow
- above ${\sim}60$ $^{\circ}\text{C},$ water can scald
- with $T_h \approx 51~^\circ\text{C}$, energy capacity ≈ 9 to 13 kWh



Three types of thermal storage

Domestic hot water use

Resistance water heaters

Heat-pump water heaters

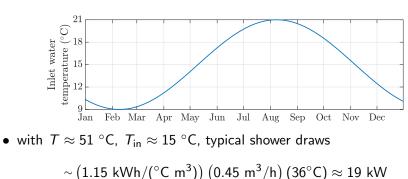
- typical household draws ~(16n + 4) gal of hot water per day
 ~16n gal for one shower or bath apiece from n occupants
 ~4 gal for everything else (like washing dishes and clothes)
- typical shower lasts ${\sim}8$ min, draws volumetric flow rate

$$\dot{V} \approx 2 \text{ gal/min} = 0.45 \text{ m}^3/\text{h}$$

Florida Solar Energy Center (2015): Estimating daily domestic hot water use in North American homes

Typical hot water thermal power draws

- thermal power associated with \dot{V} is $ho c \dot{V} (T-T_{
 m in})$
- inlet water temperature on day d of the year (d = 1 on Jan 1):



 $T_{\rm in} \approx 15$ °C + (6 °C) sin(d - 130)

Hendron (2010): Building America house simulation protocols.

Most heat withdrawals do not depend on tank temperature

- typical shower or bath has constant mixed water flow \dot{m}^{\star}
- typical bather wants constant mixed water temperature \mathcal{T}^{\star}
- SO

$$\frac{\dot{m}T + (\dot{m}^{*} - \dot{m})T_{\text{in}}}{\dot{m}^{*}} = T^{*}$$

$$\iff \dot{m}(T - T_{\text{in}}) + \dot{m}^{*}T_{\text{in}} = \dot{m}^{*}T^{*}$$

$$\iff \underbrace{\dot{m}c(T - T_{\text{in}})}_{q_{d}} = \underbrace{\dot{m}^{*}c(T^{*} - T_{\text{in}})}_{\text{independent of }T}$$

 $\implies \text{ heat withdrawal } q_d \text{ is independent of tank temperature } T$ (if T decreases, bather can mix in more hot water, less cold)

Simulating hot water draws

- showers and baths dominate hot water use
- so generate *n* showers per day, each at \sim 19 kW for \sim 10 min
- and place them at random plausible times



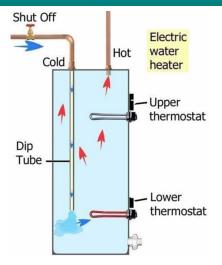
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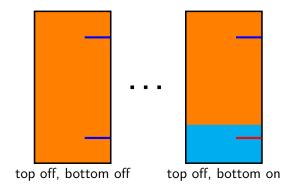
Heat-pump water heaters

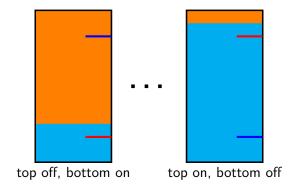
Electric resistance water heaters

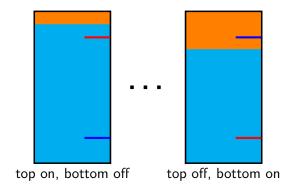


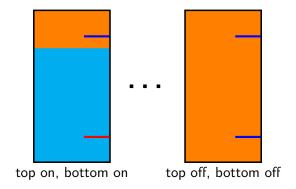
- typically designed to maintain stratification
- hot water drawn from top, cold added to bottom

- water heater controls vary by manufacturer
- one approach:
 - $\diamond\,$ one resistor near bottom, one near top
 - $\diamond~$ each resistor has a water temperature sensor
 - $\diamond\,$ run bottom resistor if bottom temperature drops below setpoint
 - $\diamond~$ switch to top resistor if top temperature drops below setpoint
 - $\diamond~$ when top temperature returns to setpoint, switch to bottom
 - $\diamond\,$ when bottom temperature returns to setpoint, turn bottom off









- anytime tank is not fully charged, exactly one resistor runs
 model:
 - \diamond try to place $x(k+1) = ax(k) + (1-a)\tau(q_c(k) + w(k))$ at \overline{x}
 - ♦ but saturate $q_c(k)$ at capacity limits $[0, \overline{p}_r]$

$$q_{c}(k) = \max\left\{0, \min\left\{\overline{p}_{r}, \frac{\overline{x} - ax(k)}{(1 - a)\tau} - w(k)\right\}\right\}$$
$$p(k) = q_{c}(k)$$



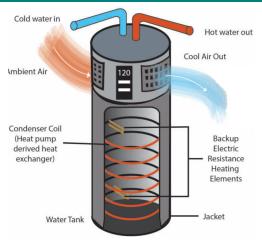
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Heat-pump water heaters (and hybrids)



- most HPWHs are hybrids with resistance backup
- heat pump COP depends on temperatures of water, air

TRC Solutions: Heat pump water heater technology

- anytime tank is not fully charged, heat pump runs
- \implies try to place x(k+1) at \overline{x}
 - but saturate $q_c(k)$ at capacity limits $[0, \eta(k)\overline{p}]$

$$egin{aligned} q_c(k) &= \max\left\{0,\min\left\{\eta(k)\overline{p},rac{\overline{x}-ax(k)}{(1-a) au}-w(k)
ight\}
ight\}\ p(k) &= q_c(k)/\eta(k) \end{aligned}$$

Hybrid water heater controls

- anytime tank is not fully charged, heat pump runs
- if charge drops below a threshold x_r, resistor also runs

\implies try to place x(k+1) at \overline{x} , but

- \diamond if $x(k) \ge x_r$, same as heat-pump-only case
- \diamond if $x(k) < x_r$, saturate $q_c(k)$ at combined limits $[0, \eta(k)\overline{p} + \overline{p}_r]$

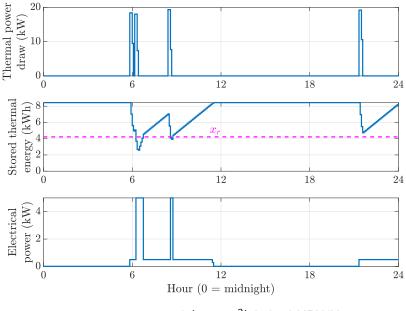
$$q_{c}(k) = \max\left\{0, \min\left\{\eta(k)\overline{p} + \overline{p}_{r}, \frac{\overline{x} - ax(k)}{(1 - a)\tau} - w(k)\right\}\right\}$$

$$p(k) = \max\left\{\frac{q_{c}(k)}{\eta(k)}, q_{c}(k) + (1 - \eta(k))\overline{p}\right\}$$

$$p\left(\frac{1}{p}\right)$$

$$\frac{p}{\eta\overline{p}}$$

$$q_{c}$$



4 occupants, 50 gal (0.19 m^3) hybrid HPWH

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